Quasi-autonomous thermal model reduction for steady-state problemsin space systems

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A matrix method is developed to reduce the number of elements of spacecraft thermal mathematical models based on the lumped parameter method. The aim of this method is to achieve a satisfactory thermal model reduction for steady-state problems, in an automatic way, while preserving the physical meaning of the system and the main characteristics of the model. The simplicity of the method, and the computational cost, are also taken into account. The reduction process is based on the manipulation of the conductive coupling matrix, which is treated as a sparse graph adjacency matrix. Then, a depth-first search algorithm is used to find the strongly connected components, which define the condensed nodes. Finally, all the thermal entities are reduced, and the results from the condensed model are compared to those from the detailed one. The entire reduction process is tested on a real thermal model, showing a good performance. In the conclusions section the characteristics and limitations of this method are shown.

1. Introduction and state of the art

Spacecraft thermal control addresses the problem of keeping space vehicles and their components temperatures within the appropriate range [1–3]. Tests and analyses are used together for the study and the design of a thermal control subsystem. In the earliest stages of the design, the preliminary studies can be performed using analytical calculations [4–9]. But as the system becomes more complex and detailed, it is necessary to rely on other tools. While the tests allow the design to be verified, they are very complex and expensive. Therefore, in recent years, thermal simulation has become more and more relevant. In Europe, the standard tool for thermal analysis in the space industry is ESATAN-TMS [10].

When working with thermal models, it is very usual to have very detailed models (for typical lumped parameter solvers, it is in the range of 10^3 – 10^4 nodes) for the element/unit which is being studied. Then, this element might have to be included in a higher hierarchy model, or it could be necessary to carry out extensive simulations (e.g. transient and sensitivity analysis). To keep the size of the model under reasonable limits ($<10^5$ nodes) and save analysis time, in all the aforementioned cases it is highly recommendable to reduce the number of nodes of the thermal model under study. The reduced thermal model must behave as the detailed one (in terms of temperatures and heat fluxes through the boundaries), while having a smaller number of nodes, so-called "size".

The classical approach to deal with this situation is to set up a reduced geometrical mathematical model (RGMM). The reduced thermal mathematical model (RTMM) is built on a user-definition basis. This means the modeler builds the RGMM, and

Nomenclature adjacency matrix, $n_d \times n_d$ Θ temperature difference matrix, $n \times n$ (K) В change of basis matrix, $n_d \times n_d$ C thermal capacity vector, $n \times 1$ (I/K) Accents specific heat capacity (J/(kg K)) derived values Ď distance matrix, $n_d \times n_d$ (m) calculated values G thermal capacity matrix, $n_d \times n_d$ (J/K) dimensionless values Н heat flux vector, $n_b \times 1$ (W) I identity matrix, $n \times n$ Subscripts K conductive coupling matrix, $n \times n$ (W/K) boundary node b L distance between node center and interface (m) C condensed model number of nodes cb conductive to boundary nodes restriction matrix, $n_d \times n_c$ Р ď detailed model dimensionless filtering threshold p_f rb radiative to boundary nodes heat flux difference vector between detailed and q sort sorted reduced model results, $n_b \times 1$ (% or W) maximum allowable heat flux difference (% or W) q_{max} Superscripts heat flux limit for separation between relative and qlim including marked boundary nodes B absolute values of heat flux difference (W) detailed model D Q thermal loads vector, $n \times 1$ (W) F filtered reduction ratio r_r including marked internal boundary nodes R radiative coupling matrix, $n \times n$ (m²) R reduced model S node cross-section (m²) S sizing connected components label array, $n_d \times 1$ S_{cn} Т transposed thickness (m) linearized T temperature vector, $n \times 1$ (K) node volume (m³) v Abbreviations w node width (m) Compressed Storage by Rows CSR X node spatial coordinates (x, y, z), $n_d \times 3$ (m) **GMM** Geometrical Mathematical Model European Space Agency **ESA** Greek symbols **FPA** Focal Plane Assembly vector of temperature differences between derived and MLI Multi-Laver Insulation calculated values (K), $n_c \times 1$ PHI Polarimetric Helioseismic Imager maximum allowable temperature difference (K) δ_{max} Reduced Geometrical Mathematical Model **RGMM** temperature difference threshold (K) ΔT_{max} **RTMM** Reduced Thermal Mathematical Model thermal conductivity (W/(m K)) 2 SCC Strongly Connected Components μ parameter (m²/s) **TMD** Thermal Model Data density (kg/m³) ρ **TMM** Thermal Mathematical Model Stefan-Boltzmann constant (W/(m2 K4)) σ

according to its definition, the nodes from the detailed thermal mathematical model (DTMM) are condensed into RTMM nodes. Then, the general practice is to get two snapshots of the DTMM (usually hot and cold operational worst cases) and, by tuning the conductive couplings and the heat loads, try to match the results from the RTMM and the DTMM. This process is time consuming, and tends to be error-prone.

Some authors and companies have developed a number of algorithms and computer codes to automatize this reduction process. For instance, Bernard et al. [11–13] have developed the TMRT (thermal model reduction tool) software. This software is intended to reduce large mathematical models. The program is based on a Guyan–Irons reduction, and it seems to work well, but it needs the reduction to be pre-defined by the engineer. This approach requires essentially defining the reduction scheme *a priori*. A very similar approach can be found in [14–16]. Deiml et al. [17] have carried out a wide analysis of the different techniques that could be applied to spacecraft thermal model reduction. They focus on nonlinear methods, which can achieve reductions with very good correlation, at the expense of losing part of the physical interpretation of the problem. They are also the only ones that also deal with the reduction of the geometrical mathematical model (GMM).

The main goals and points of the quasi-autonomous thermal reduction process presented here are:

- a. Reduce the thermal model taking into account the different quasi-isothermal zones. By grouping similar-temperature nodes, it is possible to reduce the error produced by reducing the matrices. It can be supposed that parts which are strongly connected (larger conductive couplings) will be more isothermal than other parts which are weakly connected.
- b. The model reduction should be done in a quasi-automatic way. This means, the only parameters that need to be defined are the maximum allowable temperature and heat fluxes differences, as well as the desired reduction ratio.
- c. The reduction process will depend on the temperatures and boundary conditions. Having more scenarios (e.g. hot and cold operational cases) will constrain the problem more.
- d. In engineering applications, it is essential to preserve the physical interpretation of the model elements, as well as the physical characteristics, to be able to take design decisions changing physically achievable parameters. This implies having positive thermal couplings, to preserve the boundary nodes, the heat loads, the symmetry in the coupling matrices, and the conductive physical paths.

The work presented hereafter is partially inspired by the large resistor networks reduction methods [18,19].

2. Thermal model

The lumped parameter method thermal mathematical model is described by Eq. (1), one energy balance equation per node

$$C_{i}\frac{dT_{i}}{dt} = \sum_{j=1}^{n_{d}} \left[K_{ij} (T_{j} - T_{i}) + R_{ij} (T_{j}^{4} - T_{i}^{4}) \right] + Q_{i}, \quad i = 1, \dots, n_{d}, \quad i \neq j,$$
(1)

where C_i is the thermal capacitance of node i, K_{ii} and R_{ii} the conductive and radiative couplings between nodes i and j, respectively, and Q_i the heat load on node i (R_{ii} include the Stefan-Boltzmann constant, σ). The sign criteria is $Q_i > 0$ when heat is entering the node. The coefficients of these equations are usually generated by means of a thermal analysis software package. For the development and execution of the reduction algorithm, regardless of the thermal modelling software used, all the variables which appear in (1) are considered as available inputs. These are K (conductive coupling matrix), R (radiative coupling matrix), T (temperature vector), C (node thermal capacity vector) and \mathbf{Q} (node heat load vector). Also the node spatial coordinate matrix, **X**, is considered to be known. Although the materials of the spacecraft system are obviously known, since the aim of this work is to reduce a detailed thermal model already set up, some information is considered not to be available (e.g. specific heat capacities, thermal conductivities, etc.).

3. Reduction process

3.1. Scope

Reducing linear models (those exclusively made of conductive couplings) is quite straightforward [18], whereas reducing nonlinear models (those which include radiative terms) can lead to excessive temperature deviations between the detailed and the reduced models, if the radiative terms dominate the problem. It is considered here that two nodes can be condensed only if they are connected by material medium, so that the physical meaning is preserved. That is why only the conductive couplings matrix **K** is taken into account to run the reduction algorithm. Once the model has been reduced, the radiative couplings matrix **R** is reduced using the same scheme and the problem can be linearized and solved.

A flow diagram of the reduction process that is explained in the subsequent sections is shown in Fig. 1. The goal of the reduction process is to find, in an autonomous way, the scheme that groups the thermal nodes into condensed thermal nodes. In particular, the aim is to transform the matrix system dimension from n_d (number of nodes of the detailed model) to n_c (number of nodes of the reduced model). The relation between n_d , n_c and n_b (number of boundary nodes, invariant in the reduction process) is here called reduction ratio r_r , which is a measure of the reduction degree. r_r is defined as

$$r_r = 1 - \frac{n_c - n_b}{n_d - n_b},\tag{2}$$

as boundary nodes are not considered in the reduction process. r_r ranges from 0 to 1; 0 indicates that the reduced and the detailed models are the same, whereas 1 represents complete reduction, as only the boundary nodes remain in the reduced model.

To transform the system, a matrix $\bf A$ is built which indicates the connection between nodes, taking into account the conductive coupling $\bf K$ and the temperature difference between nodes $\bf \Theta$. Matrix $\bf A$ is treated as an adjacency matrix, on which the strongly connected components (SCC) algorithm [20–22] is used. The resulting reduction scheme (represented by restriction matrix $\bf P$)

is applied to the thermal problem, reducing the dimension of matrices \mathbf{K}^R , \mathbf{R}^R , \mathbf{Q}^R . Then this system is solved, and the calculated temperatures $\widehat{\mathbf{T}}^R$ and heat fluxes $\widehat{\mathbf{H}}^R$ are compared to those derived from the detailed thermal model, $\overline{\mathbf{T}}^D$ and \mathbf{H}^D . An iteration loop is in charge of achieving the requirements imposed on temperature and heat flux differences. All the steps described above have been programmed with Python 3.4 and the Python-based scientific ecosystem SciPy. Compressed Storage by Rows (CSR) sparse matrices have been used in order to save memory and make the code more efficient.

3.2. Processing of conductive coupling matrix K

The conductive couplings between the different nodes in a thermal model are calculated based on geometrical and physical properties, as an application of Fourier's law in a lumped parameter model [23-25]. Usually, the values generated in this way are within quite a wide range (approx. 10^{-5} to 10^{+3} W/K). One of the most important contributions of this study is to realize that to compare and classify the different values as "strong" or "weak" conductors, one has to take into account not only the values themselves, but also the geometrical and physical properties which have produced such values. It should be considered that, when constructing a thermal model, different spatial scales are frequently used for different parts of the geometrical model, and they largely influence the values of the couplings. For instance, anticipating that metallic parts, such as structural panels, will be quasiisothermal, one can produce a coarser mesh. On the other hand, modelling printed circuit boards, where it could be important to distribute accurately the electric power sources, can lead to a finer mesh. This situation can produce thermal models with a great disparity of node sizes, and therefore of thermal coupling values. The diversity of bulk materials in the model construction, the simulation of thermal contacts at the bolted or glued interfaces, and other effects, greatly influence the inhomogeneity of the conductors' values. If one wants to group the nodes based on how "strong" the conductors are, it is necessary to assess the relationship between the conductor's actual value and how big this conductor could be, based on the available magnitudes. To help making this comparison, a dimensionless conductive coupling matrix, $\tilde{\mathbf{K}}$, is built.

3.2.1. Building the dimensionless coupling matrix $\widetilde{\mathbf{K}}$

To obtain the dimensionless conductive coupling matrix $\widetilde{\mathbf{K}}$ typical values of K_{ij} are needed to be used as a reference. These reference values are grouped inside the conductive coupling sizing matrix, $\mathbf{K^S}$. First, let us build the conductive coupling sizing matrix, $\mathbf{K^S}$. As we want to identify which nodes should be condensed, we are interested in having orders of magnitude for the conductive couplings in $\mathbf{K^S}$ based on the size of the contact and the materials involved. This matrix defines a potential value for a typical coupling for each conductive coupling existing in the model. For each non-zero ij element of matrix \mathbf{K} , the conductive coupling between node i and j is defined as follows

$$K_{ij}^s = \frac{1}{\frac{L_i}{\lambda_i S_i} + \frac{L_j}{\lambda_i S_i}},\tag{3}$$

where λ_i and λ_j are the thermal conductivities, L_i and L_j the distances between the node centers and the interface, and S_i and S_j the cross section for the conductive coupling calculation. All the geometric magnitudes involved are shown in Fig. 2.

The thermal conductivities λ_i are not explicitly included in the inputs mentioned in Section 2, so they are unknown. To estimate the sizing matrix $\mathbf{K}^{\mathbf{S}}$, a reference value, say λ , is used for all the conductors. This generic value itself does not matter and does not

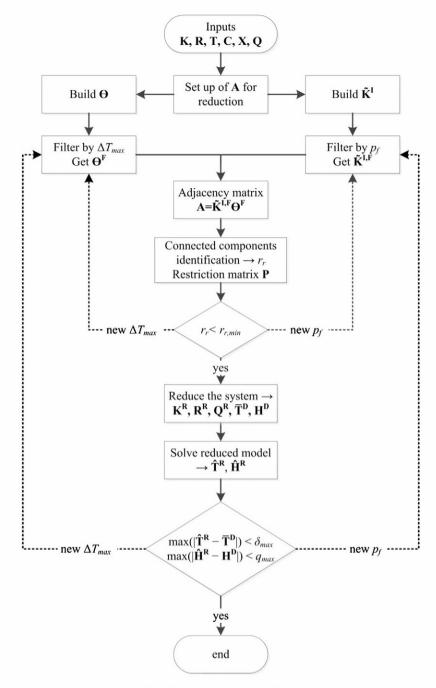


Fig. 1. Reduction process flow diagram.

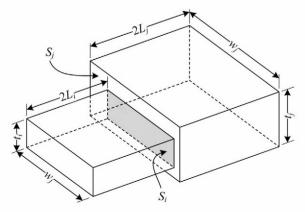


Fig. 2. Conductive coupling calculation scheme.

influence the results. It is intended just to obtain values with thermal engineering meaning. Therefore

$$K_{ij}^s = \frac{\lambda}{\frac{L_i}{S_i} + \frac{L_j}{S_j}}.$$
 (4)

 S_i, S_j are also unknown, and estimated as follows. Let us consider the thermal capacities for the nodes i and j

$$\begin{cases}
C_i = \rho_i c_{p_i} v_i = \rho_i c_{p_i} w_i t_i 2L_i \\
C_j = \rho_j c_{p_i} v_j = \rho_j c_{p_j} w_j t_j 2L_j
\end{cases}$$
(5)

where, as for the thermal conductivities, ρ and c_p cannot be inferred from the input data. Again, standard values are used for these magnitudes. S_i and S_j can be defined as

$$S_i = w_i t_i; \quad S_j = w_j t_j. \tag{6}$$

Although w and t are unknown magnitudes, they can be obtained from (5). Isolating the group $w \cdot t$, one gets

$$w_i t_i = \frac{C_i}{2\rho c_p L_i}; \quad w_j t_j = \frac{C_j}{2\rho c_p L_i}. \tag{7}$$

Substituting (6) and (7) into (4)

$$K_{ij}^{s} = \frac{\lambda}{\frac{L_{i}}{w_{i}t_{i}} + \frac{L_{j}}{w_{j}t_{j}}} = \frac{\lambda}{2\rho c_{p}} \cdot \frac{1}{\frac{L_{i}^{2}}{C_{i}} + \frac{L_{j}^{2}}{C_{i}}}.$$
 (8)

We can try to estimate the values for L_i and L_j . From **X** the distance between nodal centers i and j, D_{ij} , can be calculated. It is assumed that the nodal geometries are similar, and therefore the distances L_i and L_j are of the same order of magnitude. Then

$$L_i \sim L_j \to L_i = L_j = \frac{D_{ij}}{2}. \tag{9}$$

And consequently, using (8) and (9) we get

$$K_{ij}^{s} = \frac{2\lambda}{\rho c_{p}} \cdot \frac{1}{D_{ij}^{2}} \cdot \frac{1}{\frac{1}{C_{i}} + \frac{1}{C_{i}}} = \mu \cdot \frac{G_{ij}}{D_{ij}^{2}}, \tag{10}$$

where the parameter μ is defined as

$$\mu = \frac{2\lambda}{\rho c_n},\tag{11}$$

and

$$G_{ij} = \frac{1}{\frac{1}{C_i} + \frac{1}{C_j}}. (12)$$

The sizing matrix $\mathbf{K}^{\mathbf{S}}$ is obtained from (8) by calculating the thermal capacity and distance matrices, \mathbf{G} and \mathbf{D} , which are symmetric, together with the parameter μ . The choice of the parameter μ does not affect the construction of the dimensionless matrix $\widetilde{\mathbf{K}}$, as the same value is used for all the elements. But it is important to choose a realistic value, so that the sizing conductive coupling matrix $\mathbf{K}^{\mathbf{S}}$ has realistic values as well. Assuming typical values, such as $\lambda = 50 \, \mathrm{W/(m \, K)}$, $\rho = 3000 \, \mathrm{kg/m^3}$ and $c_p = 1000 \, \mathrm{J/(kg \, K)}$, these would yield a value for $\mu = 1.67 \cdot 10^{-5} \, \mathrm{m^2/s}$.

Once the sizing matrix K^s has been calculated, the dimensionless matrix \widetilde{K} is obtained by dividing element-wise the matrix K by K^s (only applies for non-zero elements)

$$\widetilde{K}_{ij} = \frac{K_{ij}}{K_{ii}^S}, \quad i, j : K_{ij} \neq 0, \quad i \neq j.$$

$$(13)$$

 $\tilde{\mathbf{K}}$ represents the comparison between the conductive couplings of the detailed model and their estimated values obtained from the available data, such as geometry and thermal capacities.

3.2.2. Boundary nodes isolation

The boundary nodes have to be retained in the reduced model, and should not be condensed. They should remain as independent nodes, that is, "isolated". Therefore, they have to be artificially decoupled from the rest of the nodes in the model in order to process the adjacency matrix \mathbf{K} need to be marked. Basically, this means that all the related couplings to the boundary nodes will be artificially set to zero, so that they are not able to be condensed with any other node (the boundary nodes appear as if they were isolated). It is important to underline that this is done only in the $\widetilde{\mathbf{K}}$ matrix (the resulting matrix is called $\widetilde{\mathbf{K}}^{\mathbf{B}}$). It only affects the reduction scheme, but all the information regarding the conductive couplings between the boundary nodes and rest of the model is still available, and is consequently included in the reduced model.

In order to improve the processing of the conductive coupling matrix and the outcome of the model reduction, it is useful to isolate the nodes which are directly conductively coupled to the boundary nodes. These nodes are called "internal boundary nodes". Regardless of the aim of the model reduction (intensive internal calculations, delivery to an assembler of thermal models, etc.), the isolation of these internal nodes will generally allow for a better temperature correlation to be achieved in these nodes, which ultimately will yield a better heat fluxes correlation (because the conductive couplings between the boundary nodes and the internal boundary nodes are kept independent from the general reduction process). The process is applied in \widetilde{K}^B , and the resulting matrix is called \widetilde{K}^I .

Regarding the boundary nodes that are connected to the rest of the model only by means of radiative couplings, they are isolated as well, as their conductive couplings with the model are obviously zero.

3.2.3. Filtering of the dimensionless conductive coupling matrix

The matrix $\widetilde{\mathbf{K}}^1$ is ready to be filtered by a value p_{f_i} which represents a dimensionless threshold as follows

$$\widetilde{K}_{ij}^{l,F} = \begin{cases} 1 & \widetilde{K}_{ij}^{l} > p_{f} \\ 0 & \widetilde{K}_{ij}^{l} \leqslant p_{f} \end{cases}$$
 (14)

The resulting matrix $\widetilde{\mathbf{K}}^{\mathrm{LF}}$ is "binarized", a matrix with only ones and zeros, which contains the information of the nodes that are to be merged. The number of condensed nodes depends on p_f . $\widetilde{K}_{ij}^{\mathrm{LF}} = 1$ means a good connection, and therefore the connected nodes have to be included in the same condensed node. As the threshold value p_f increases, more nodes appear as isolated in the model, and consequently the number of nodes in the reduced model is larger.

3.3. Construction and process of temperature difference matrix

The representativeness of the reduced model is evaluated by comparing the temperatures of the reduced model and of the detailed model. Therefore, if the nodes that form a condensed node are isothermal (or nearly isothermal), the reduced model will behave more similarly to the detailed one. Thus, it is reasonable to limit the temperature difference that two nodes can have in order for them to be included in the same condensed node.

3.3.1. Building and filtering the temperature difference matrix

The temperature difference matrix $\boldsymbol{\Theta}$ is built with the nodal temperature vector \boldsymbol{T} of the detailed model.

$$\Theta_{ij} = |T_i - T_j|. \tag{15}$$

The matrix Θ is ready to be filtered by a maximum temperature difference, ΔT_{max} . The resulting matrix is "binarized" in the same fashion as explained in Section 3.2.3, as follows,

$$\Theta_{ij}^{F} = \begin{cases} 0 & \Theta_{ij} > \Delta T_{max} \\ 1 & \Theta_{ij} \leqslant \Delta T_{max} \end{cases}$$
 (16)

Obviously, the greater the maximum temperature difference ΔT_{max} , the lower the number of nodes in the reduced model.

3.4. Model condensation

3.4.1. Definition of the adjacency matrix

With $\widetilde{\mathbf{K}}^{I,F}$ and $\mathbf{\Theta}^F$ already built, it is necessary to merge both matrices in the adjacency matrix \mathbf{A} . This is achieved by multiplying both matrices element-wise

$$A_{ij} = \widetilde{K}_{ij}^{I,F} \cdot \Theta_{ij}^{F}, \tag{17}$$

where basically we are restricting the filtered dimensionless matrix $\widetilde{\mathbf{K}}^{\mathbf{I},\mathbf{F}}$ with $\mathbf{\Theta}^{\mathbf{F}}$. Because of the definition of $\widetilde{\mathbf{K}}^{\mathbf{I},\mathbf{F}}$ and $\mathbf{\Theta}^{\mathbf{F}}$ in Eqs. (14) and (16), the adjacency matrix \mathbf{A} will have only zeros and ones. It represents the thermal network as a graph, the nodes being the graph vertices and the conductive couplings the edges. The thermal network is an undirected graph, and \mathbf{A} is a sparse adjacency symmetrical matrix. As mentioned in Section 3.1, the programming has been carried out using sparse matrices in CSR format [26].

3.4.2. Connected components algorithm

The model is reduced by running the connected components algorithm [20–22], so that the nodes are grouped according to their connections (the algorithm finds the nodes which are connected to each other). As a result, the algorithm returns the number of condensed nodes n_c , and the connected component labels array $\mathbf{S}_{\mathbf{cn}}$, in which the nodes belonging to the same connected component (thus, belonging to the same condensed node) have the same identifier.

3.4.3. Sorting the matrix entities

Sorting the original K and R matrices is not strictly necessary for the reduction process. However, arranging the thermal coupling matrices by condensed nodes order can give visual information about how the reduction process has grouped the nodes. To do this, a change of basis matrix B is necessary, which is built with the label array S_{cn} . Then, one can get the sorted versions of K and R as the following matrix products

$$\begin{aligned} & K_{sort} = B \cdot K \cdot B^{T} \\ & R_{sort} = B \cdot R \cdot B^{T} \end{aligned} \tag{18}$$

(See example in Section 4.1).

3.4.4. Compacting the model

To reduce the dimension of the input matrices and vectors a change of basis matrix, which is called restriction matrix \mathbf{P} , is needed. The dimensions of \mathbf{P} are n_d rows \times n_c columns. The node order of \mathbf{P} being the same as in \mathbf{K} , each row corresponds to a node, which will have a "1" in the column corresponding to the associated condensed node, and the rest of the elements in the row will be "0". With the restriction matrix, and the original inputs, the matrix entities can be built for the reduced model

$$K^{R} = P^{T} \cdot K \cdot P
R^{R} = P^{T} \cdot R \cdot P$$
(19)

Thus, for each condensed node, all of its conductive couplings with a neighbor are summed up. Same methodology is used for the radiative couplings. Actually, these combinations are the source of the deviation between the condensed nodes predicted temperatures (derived from the detailed thermal model) and those calculated by solving the reduced thermal model. The errors will be small as far as the condensed nodes are isothermal. Thus, the threshold values (p_f and ΔT_{max}) determine the error obtained in the reduction, considering the rest of the elements constant.

Also for the loads and thermal capacities vectors

$$\begin{aligned} \boldsymbol{Q}^{R} &= \boldsymbol{P}^{T} \cdot \boldsymbol{Q} \\ \boldsymbol{C}^{R} &= \boldsymbol{P}^{T} \cdot \boldsymbol{C} \end{aligned} \tag{20}$$

This operation assigns to each condensed node the sum of the thermal capacities of the nodes that belong to that condensed node. The same occurs for the thermal loads. The reduced system can be solved with the reduced matrices and vectors to obtain $\widehat{\mathbf{T}}^{\mathbf{R}}$. However, the correctness or accuracy of the reduction process needs to be evaluated. Thus, this matrix needs to be compared with

the target temperatures \overline{T}^p , derived directly from the detailed model. One widely accepted standard in the industry is to calculate these temperatures as the thermal capacity weighted mean temperature values (it is also possible to weight the temperatures with nodal areas)

$$\overline{T}_{i}^{D} = \frac{\sum_{j=1}^{m} T_{j} C_{j}}{\sum_{i=1}^{m} C_{i}}, \quad i = 1, \dots, n_{c},$$
(21)

where m is the number of nodes that belong to condensed node i. Once the reduced model is solved, the temperatures derived from the detailed model, $\widehat{\mathbf{T}}^{\mathbf{p}}$, are compared with those calculated with the reduced model, $\widehat{\mathbf{T}}^{\mathbf{R}}$. The boundary nodes are directly assigned a temperature in the reduced model, as they were isolated in the adjacency matrix (Section 3.2.2).

Regarding the heat fluxes, the calculated values for the reduced model, \widehat{H}^R , are compared with the target values, those corresponding to the detailed model, H^D_{cb} and H^D_{rb} . H^D_{cb} accounts for the conductive heat fluxes to boundary nodes, whereas the vector H^D_{rb} does so for the radiative heat fluxes.

3.5. Validation of the reduced thermal model

With the reduced matrices and vectors, the reduced system can be solved. $\overline{\mathbf{T}}^0$ is used to set up the boundary conditions, and also as an initial condition if an iterative solver is employed. The thermal system is then solved in steady state conditions, one energy balance equation per node

$$\sum_{j=1}^{n_c} \left[K_{ij}^R \left(\widehat{T}_i^R - \widehat{T}_j^R \right) + R_{ij}^R \left(\widehat{T}_i^{R^4} - \widehat{T}_j^{R^4} \right) \right] + Q_i^R = 0, \quad i = 1, \dots, n_c, \quad i \neq j.$$
(22)

The model is solved with a steady state solver. For the linearization of the radiative terms, the following algebraic identity is used (see [27])

$$\left(\widehat{T}_{i}^{4}-\widehat{T}_{j}^{4}\right)=\left(\widehat{T}_{i}^{2}+\widehat{T}_{j}^{2}\right)\left(\widehat{T}_{i}+\widehat{T}_{j}\right)\left(\widehat{T}_{i}-\widehat{T}_{j}\right). \tag{23}$$

Using Eq. (23) in the non-linear part of Eq. (22), one gets

$$R_{ij}^{R}\left(\widehat{T}_{i}^{4}-\widehat{T}_{j}^{4}\right)=R_{ij}^{R}\left(\widehat{T}_{i}^{2}+\widehat{T}_{j}^{2}\right)\left(\widehat{T}_{i}+\widehat{T}_{j}\right)\left(\widehat{T}_{i}-\widehat{T}_{j}\right)=K_{ij}^{R^{*}}\left(\widehat{T}_{i}-\widehat{T}_{j}\right), \quad (24)$$

where

$$K_{ij}^{R^*} = R_{ij}^R \left(\widehat{T}_i^2 + \widehat{T}_j^2 \right) \left(\widehat{T}_i + \widehat{T}_j \right). \tag{25}$$

The values for $\mathbf{K^{R'}}$ are obtained with the temperatures calculated for the current step. Then, the system is solved, and the latest temperatures are compared with those corresponding to the previous step. Once the model has been solved, the results must be compared with those coming from the detailed model. For the temperatures, the absolute difference will have to be less than a predefined value, δ_{max} . Hence, with the reduced model derived temperatures $\overline{\mathbf{T^{D}}}$, and the reduced model calculated temperatures $\widehat{\mathbf{T^{R}}}$, it can be checked that the difference is below the maximum allowable value δ_{max} .

$$\delta_{i} = \left| \overline{T}_{i}^{D} - \widehat{T}_{i}^{R} \right| < \delta_{max}, \quad i = 1, \dots, n_{c}.$$
 (26)

Regarding the heat fluxes exchanged with the boundary nodes, the heat fluxes vectors will have as many components as the number of boundary nodes, n_b . The difference can be expressed in terms of percentage or in absolute numbers, depending on the heat flux value (the limit is defined by q_{lim}). For the conductive heat fluxes

$$\begin{cases} \tilde{q}_{cb_{i}} = \left| \frac{H_{cb_{i}}^{D} - \widehat{H}_{cb_{i}}^{R}}{\min \left\{ H_{cb_{i}}^{D} - \widehat{H}_{cb_{i}}^{R} \right\}} \right| < \tilde{q}_{max}, & q_{cb_{i}} \geqslant q_{lim}, \\ q_{cb_{i}} = \left| H_{cb_{i}}^{D} - \widehat{H}_{cb_{i}}^{R} \right| < q_{max} \ W, & q_{cb_{i}} < q_{lim}, \end{cases}$$
 $i = 1, \dots, n_{b}, \quad (27)$

and for the radiative heat fluxes

$$\begin{cases} \tilde{q}_{rb_{i}} = \left| \frac{H_{rb_{i}}^{D} - \widehat{H}_{rb_{i}}^{R}}{\min \left\{ H_{rb_{i}}^{D} \cdot \widehat{H}_{rb_{i}}^{R} \right\}} \right| < \tilde{q}_{max}, & q_{rb_{i}} \geqslant q_{lim}, \\ q_{rb_{i}} = \left| H_{rb_{i}}^{D} - \widehat{H}_{rb_{i}}^{R} \right| < q_{max} W, & q_{rb_{i}} < q_{lim}, \end{cases}$$
 $i = 1, \dots, n_{b}.$ (28)

4. Test model

In order to analyze the application of the method to a real problem with a large number of nodes, let us consider the thermal model of the Focal Plane Assembly (FPA) of the Polarimetric and Helioseismic Imager (PHI) instrument. PHI will be on board the ESA mission Solar Orbiter, to be launched in 2018 [28]. The FPA detailed thermal model consists of 1072 nodes and 457 geometric primitives, and has been entirely set up with ESATAN-TMS [10]. The FPA is a camera system, with a detector, front-end electronics, and a cold finger which connects the camera to the PHI Optics Unit cold element by means of a thermal strap. It also has an aluminum

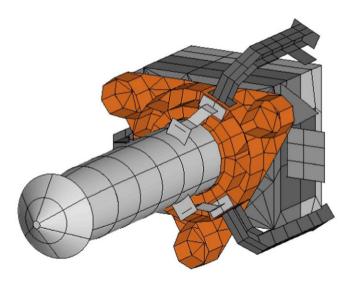


Fig. 3. PHI Optics Unit FPA GMM.

housing, and an aluminum tube with some lenses. The PHI FPA geometrical mathematical model (GMM) is shown in Fig. 3.

4.1. Reduction of the thermal mathematical model

The conductive and radiative coupling matrices are shown in Fig. 4 as imported from the ESATAN-TMS TMD (Thermal Model Data) file. Matrix **K** has 2367 conductive couplings, whereas **R** has 299,252. The matrix **R** is very dense not only because it represents the radiative couplings, but also due to the accuracy parameters used in the radiative calculations. The matrices are symmetric, so the program only works with upper triangular matrices. All the graphics related to the reduction process have been done with Matplotlib [29].

The model has four boundary nodes. If they are isolated, and the strongly connected components algorithm on the resulting matrix $\mathbf{K}^{\mathbf{B}}$ is run, the minimum number of condensed nodes is found to be 6 (including the 4 boundary nodes and 2 nodes which are decoupled from each other). The maximum ratio r_r , which can be got according to Eq. (2), is 0.998 (from 1072 to 6). This is a theoretical limit, but in order to correlate the models, this value of r_r will have to be lower.

To prepare the matrix **K** for the filtering, μ has been given the value calculated in Section 3.2.1 (μ = 1.67 · 10⁻⁵ m²/s). Then, with μ , together with the distance matrix **D** and the capacitance matrix **G**, the sizing matrix **K**^S can be obtained. The distribution of the conductive couplings matrices **K** and **K**^S is shown in Fig. 5.

In Fig. 5, it can be seen that most of the couplings for matrix ${\bf K}$ are concentrated in the range 10^{-1} – 10^0 W/K, whereas for matrix ${\bf K}^{\bf S}$ they are mostly concentrated in the range 10^1 – 10^2 W/K. This means that, according to the value chosen for parameter μ and the magnitudes used to estimate ${\bf K}^{\bf S}$, the model has a "poor" conductive behavior (λ has a generic value of 50 W/(m K)). This appraisal would change in the moment μ is changed, but still it gives an idea of how conductive the model is if it is compared with the estimator.

Once the dimensionless coupling $\mathbf{K}^{\mathbf{I}}$ and temperature $\mathbf{\Theta}$ matrices have been created, it is possible to set up the adjacency matrix \mathbf{A} , filter the system and compact the model. Before that, it was decided to do a preliminary study regarding how the reduction ratio r_r varies with the different filtering levels. Fig. 6 shows the variation of r_r with p_f and ΔT_{max} independently. This figure confirms what was explained in Sections 3.2.3 and 3.3.1 relative to the behavior of r_r as a function of p_f and ΔT_{max} .

As the reduction ratio r_r is a function of both p_f and ΔT_{max} , to analyze the influence of these two parameters an appropriate method is to create a contour plot of the function $r_r = f(p_f, \Delta T_{max})$,

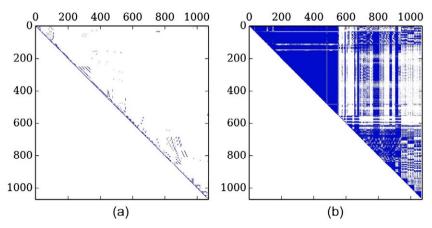


Fig. 4. Sparse graphical representation of (a) matrix K and (b) matrix R.

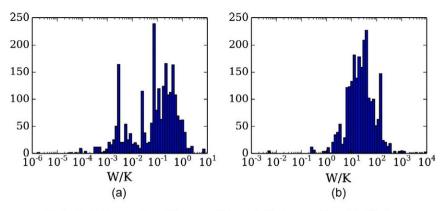


Fig. 5. Conductive couplings histograms of (a) matrix K values and (b) matrix K^S values.

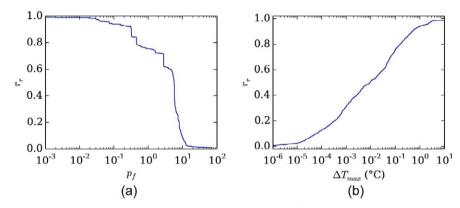


Fig. 6. Variation of (a) r_r vs. p_f , and variation of (b) r_r vs. ΔT_{max} . (a) The SCC algorithm is run with $\widetilde{\mathbf{K}}^{\mathrm{IF}}$. (b) The matrix \mathbf{K} is multiplied element-wise by $\mathbf{\Theta}^{\mathrm{F}}$, and the SCC algorithm is run with the resulting matrix.

as shown in Fig. 7. This way, a combination of p_f and ΔT_{max} suitable to get the desired value of r_r can be selected. Furthermore, it is necessary to determine if the model correlation is good enough. To do this, a correlation criteria for the reduced model should be defined.

For the sake of simplicity, only one is used for all the parts: δ_{max} = 3 K. Regarding the heat flux exchanged with the boundary nodes, the chosen criteria are:

- a. For absolute heat flow less than $q_{\it lim}$ = 1 W, difference must be less than 0.1 W.
- b. For absolute heat flow more than q_{lim} = 1 W, difference must be less than 10%.

Therefore, the value for q_{max} is 0.1 and \tilde{q}_{max} is 10%. Fig. 7 shows the function $r_r = f\left(p_f, \Delta T_{max}\right)$, together with the correlation status for each pair of values p_f and ΔT_{max} . The successful correlation implies that the three conditions (temperature difference, conductive and radiative heat fluxes differences) are satisfied. The values for p_f and ΔT_{max} cover almost the full range of both magnitudes, as shown in Fig. 6. The most extreme values have been omitted. As can be seen in Fig. 6, the extreme values of the variation range for p_f and ΔT_{max} produce almost no change in the reduction ratio r_f .

Hence, it can be seen from the results shown in Fig. 7 that it is not possible to have a reduction ratio r_r of more than 0.75–0.80 that yields a successful correlation (with the current correlation criteria). This is due to the PHI FPA model design idiosyncrasy. Other models could yield greater values of r_r that satisfy the correlation criteria. In order to study in detail this model reduction in particular, let us plot the same Fig. 7 but zooming in on the range of r_r = [0.50,0.90]. This zooming in, together with the display of temperature and heat flux difference maps, is shown in Fig. 8.

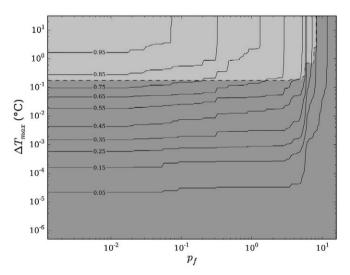


Fig. 7. Contour plot of the function $r_r = f(p_f, \Delta T_{max})$. Black solid lines represent constant r_n whose value is indicated inline. Lighter gray zone indicates successful correlation for the resulting reduced model. Darker gray colored zone indicates failed correlation for the resulting reduced model. Dashed black line indicates the separation between both zones, being effectively the limit for a successful correlation.

As shown in Fig. 6, the variation of r_r with p_f is steep, whereas the variation of r_r with ΔT_{max} is more gradual. This behavior can be observed in Fig. 7. For the same value of ΔT_{max} and r_r , it is better to choose a higher value of p_f , as the potential number of conductive couplings will be lower. We can also see that the most restric-

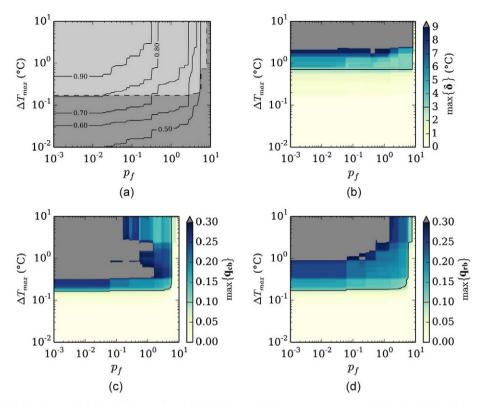


Fig. 8. (a) Contour plot of the function $r_r = f(p_f, \Delta T_{max})$. Zoom at the $r_r = [0.5, 0.9]$ zone. (b) Covers the same area as (a) but the displayed magnitude is max{δ}. Black line delimit the successful correlation zone in terms of temperature difference. (c) Covers the same zone as (a) and (b), but with regard to max{ $\mathbf{q_{cb}}$ }. The same area policy applies to (d), but in this case the plotted magnitude is max{ $\mathbf{q_{rb}}$ }.

tive criteria for the correlation are the heat flux differences (except for a small area where $p_f \approx 10^1$ and ΔT_{max} ranges between 10^0 and 10^1 °C).

Let us select a combination of values for p_f and ΔT_{max} : p_f = 0.1 and ΔT_{max} = 0.15 °C. After applying the SCC algorithm, these values yield a n_c = 272 nodes. With n_b = 4, Eq. (2) gives r_r = 0.75.

The outcome of the SCC algorithm allows the restriction matrix P to be built (Fig. 9), as well as the sorted matrices K_{sort} and R_{sort} (Fig. 10). With the restriction matrix P the reduced matrices K^R and R^R can be built (Fig. 11).

With the values of p_f and ΔT_{max} chosen, it is already known by looking at Fig. 8, that the correlation of the reduced model with the detailed model will be within the limits. The reduced system to find out the exact values for the temperature and heat fluxes differences can now be solved. Fig. 12 shows the components for the temperature difference vector δ . As can be deduced from this figure, the maximum value is around 0.35 K, and the minimum is around -0.25 K.

Regarding the conductive heat flux differences, the values from the detailed model H^D_{cb} and those calculated with the reduced model \widehat{H}^R_{cb} are

$$\begin{cases} \mathbf{H_{cb}^{D}} = [2.89 \quad 0.67 \quad 0.05 \quad 0.00] \quad (W) \\ \widehat{\mathbf{H}_{cb}^{R}} = [2.89 \quad 0.73 \quad 0.04 \quad 0.00] \quad (W) \end{cases}$$
 (29)

If both vectors are compared, it can be seen that the maximum difference occurs for the second boundary node. According to the definition given in Eq. (27), together with our criteria about heat flux differences, it can be seen that the goal value is smaller than 1 W, and consequently the difference must be given in absolute values (and not in percentage),

$$q_{cb_2} = \left| H_{cb_2}^D - \widehat{H}_{cb_2}^R \right| = 0.06 \text{ W} < q_{max} = 0.1 \text{ W},$$
 (30)

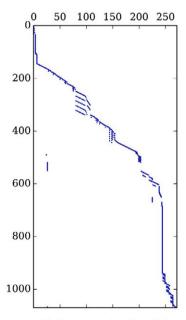


Fig. 9. Sparse graphical representation of restriction matrix P.

and for radiative heat fluxes

$$\begin{cases} \mathbf{H_{rb}^{D}} = \begin{bmatrix} 0.00 & 0.00 & 0.00 & -1.17 \end{bmatrix} & (W) \\ \widehat{\mathbf{H}_{rb}^{R}} = \begin{bmatrix} 0.00 & 0.00 & 0.00 & -1.22 \end{bmatrix} & (W) \end{cases}$$
 (31)

where the difference, according to Eq. (28) is

$$\tilde{q}_{rb_4} = \left| \frac{H_{rb_4}^D - \hat{H}_{rb_4}^R}{\min \left\{ H_{rb_4}^D, \hat{H}_{rb_4}^R \right\}} \right| = 4.3\% < \tilde{q}_{max} = 10\%.$$
(32)

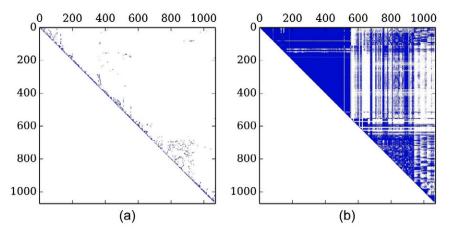


Fig. 10. Sparse graphical representation of (a) matrix K_{sort} and (b) matrix R_{sort} . The matrices are symmetric, so the program only works with upper triangular matrices.

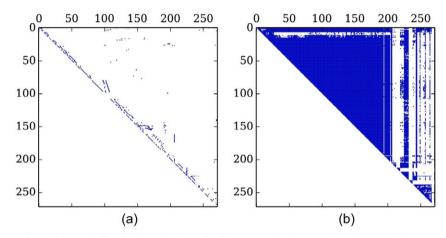


Fig. 11. Sparse graphical representation of (a) matrix K^R and (b) matrix K^R has 513 conductive couplings, whereas R^R has 23,917. The matrices are symmetric, so the program only works with upper triangular matrices.

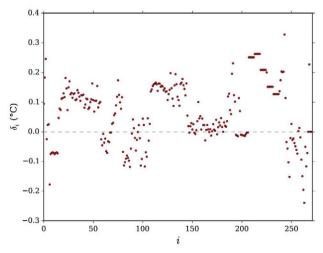


Fig. 12. Components of vector δ .

The program takes 180 min in a mid-2014 desktop computer to run 25,350 cases (reduced models obtained with different combinations of p_f and ΔT_{max}), in order to plot Figs. 7 and 8.

4.2. Reduction of the geometrical mathematical model

Based on the reduction scheme developed in Section 4.1, it is necessary to reduce the GMM accordingly. Although it is possible

to develop a kind of automatic algorithm to reduce automatically the GMM [17], we believe that the manual reduction serves its purpose, and it is not as time consuming as the TMM reduction. With the adequate tools to get the information from the TMM reduction scheme, the GMM reduction can be very effective.

5. Conclusions

The method presented in this paper works well with a broad selection of thermal models. It is focused on internally mounted space scientific instrument thermal models, although formally the method could be used in anyone. The method uses the steady-state conditions for the reduction process, finding the condensed nodes based on thermal conductance values and temperature similarities. It is fast, intuitive and preserves all the physical characteristics of the original thermal model. The value of the maximum reduction ratio r_r for which the correlation is successful depends on the model, but, according to the tests carried out, it seems quite difficult to achieve reductions with a reduction ratio of 90% or more. If it is necessary to reach higher levels of reduction, then one option could be to use optimization and stochastic methods on top of the reduction method explained in this paper [30–32].

The transient correlation is beyond the scope of this study, but first studies show that the reduced models obtained by this technique behave reasonably well. Other aspects which have not been dealt with here are multi-layer insulation (MLI), thermo-electric coolers and other special thermal control technologies models reduction.

Acknowledgements

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